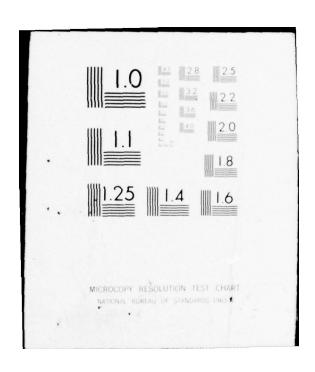
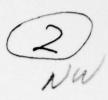
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NAVAL POSTGRADUATE SCHOOL

Monterey, California



THESIS

POWER OPTIMIZATION OF THE CAPTURED AIR BUBBLE SURFACE EFFECTS SHIP

by

Frederick Kenneth Richardson December, 1976

Thesis Advisor:

G. J. Thaler

Approved for public release; distribution unlimited





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POWER CPTIMIZATION OF THE CAPTURED AIR BUBBLE SURFACE EFFECTS SHIP

by

Frederick Kenneth Richardson Lieutenant, United States Navy B.S., Purdue University, 1968

Submitted in partial fulfillment of the requirements for the degree of

MASTER OF SCIENCE IN ELECTRICAL ENGINEERING

from the NAVAL POSTGRADUATE SCHOOL December, 1976

Approved by:

Chairman, Department of Electrical Engineering

Mark Omman

Dean of Science and Engineering

ABSTRACT

Through the use of simulation studies of the Surface Effects Ship (SES) XR-3, it is shown that power optimization can be achieved by controlling the air bubble plenum pressure and the pitch angle of the craft. Studies indicate a savings of up to forty percent in total power required for cruising speeds in the range of fifteen to thirty knots.

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I. INTRODUCTION

A. BACKGROUND

The conventional displacement vessel exhibits a well known and documented speed limitation caused by drag characteristics of the hull-water interface. In an effort to effect a great increase in surface vessel speed, a program has been initiated by the United States Navy to develop various craft whose principal means of support is other than hydrostatic lift.

One such type of craft currently receiving attention is the Surface Effect Ship (SES). There are basically two types of ships in this category, the Air Cushion Vehicles, or howercraft, and the Captured Air Bubble (CAB) craft. The general nature of these craft and their construction is well presented by Robert L. Trillo in Reference 1. Either all or a major portion of the craft support is obtained from a pressure differential between the atmosphere and a plenum chamber which is open at the bottom. The great speed advantages of the SES are from two principal characteristics: (1) energy is not wasted by displacing a large volume of water, and (2) the frictional forces at the hull-water interface are greatly reduced by keeping the structure actually in water contact to a minumum.

Surface Effects Ships are generally categorized as "Air Cushion Vehicles" whose weight is entirely supported by the pressure differential in the plenum chamber or "Captured Air

Bubble Craft" whose weight is partially supported by a sidewall structure which extends into the water. For United States Navy applications, the Captured Air Bubble (CAB) craft is being researched.

The term Captured Air Bubble is slightly misleading since the plenum chamber air does leak out and thus must be continuously. replenished by supply fans. When compared to the air cushion vehicle, however, this leakage rate is relatively small. The Air Cushion Vehicle has a continuous gap around its entire periphery, whereas the Captured Air Bubble craft has leakage only from the stern seal, thus the plenum chamber supply fans of the Air Cushion Vehicle must be much larger and more powerful than those of the CAB of similar size.

This thesis is concerned with simulation studies of the Captured Air Bubble craft utilizing a digital computer, specifically the Loads and Motions Program developed by Oceanics, Incorporated.

The basic rigid body analysis and spatial relationships of the Loads and Motions Program are well documented in Reference 2, and thus will not be duplicated here. The principal static and dynamic approximations used in developing the equations of motion for the craft in its six degrees of freedom are also covered.

The Loads and Motions Program has been converted to represent the Naval Postgraduate School's SES test craft, the XR-3. All simulation studies for this thesis were accomplished utilizing the XR-3 Loads and Motions simulation program.

B. OBJECTIVES

The purpose of this thesis is to take a detailed look into the aspects of power minimization at various cruising speeds in both calm water and sea state conditions. Pitch angle was utilized to reduce hull drag effects and introduce planing action, while lift fan speed was varied to control the air cushion bubble pressure, and thus the draft of the craft. The results are presented in both tabular and graphical form. The results are also shown in the form of recommended operating profiles.

II. GENERAL DISCUSSION

A. INTRODUCTION

Previous studies indicate that significant performance benefits can be obtained by controlling the pressure of the air bubble which supports the craft. It is clear that low bubble pressures would require large thrust values to maintain a given speed primarily because of the greater wetted surface at the hull-water interface causing increased drag forces. If the bubble pressure is increased, the draft decreases and it is expected that the thrust required to maintain that speed to decrease, but at the same time the fan power required to support the craft will increase. Intuitively, one expects that the total power (Thrust Power + Fan Power) will reach a minumum at some operating point. It is the purpose of this thesis to investigate and determine that operating point.

Additionally, it is found that the thrust power required varied as a function of the pitch angle of the craft. One might now ask the following questions:

- 1. Is there a global minimum to be found?
- 2. Does a change in fan power have a significant effect on total power?
- 3. Is the pitch angle a significant factor in controlling thrust power?

All studies were initially conducted for calm water conditions. Six different speeds in the cruising range of fifteen to thirty knots were studied extensively. Nine plenum bubble pressures were utilized at each speed to obtain a family of curves for analysis. After the calm water simulations were complete, the XR-3 craft was operated in calm water to verify the trends found in the computer simulation. Additionally, two speeds were chosen, eighteen and twenty-seven knots, for sea state simulation studies. Three plenum pressures were utilized to check for correlation between calm water and sea state operation and to generate a set of curves for comparison.

B. SIMULATION METHODS

Simulation was achieved by utilizing the existing six degree of freedom simulation model program for the 100-B surface effects ship as modified for the XR-3 craft. This program has undergone exhaustive analysis at the Naval Postgraduate School to determine its accuracy in predicting craft behavior and it is felt to be adequate for this study (References 3, 4 and 5). The basic program was modified slightly to obtain the output of data necessary for the completion of this study. The constant input parameters were also changed to reflect recent modifications to the craft seals and appendages.

The actual weight distribution of the XR-3 craft is not presently known exactly, so an approximation was determined by an iterative method. By a simulation program, several masses were moved about the craft until the same magnitude of moments about the X, Y and Z axes were obtained as had been utilized in previous studies of the craft. This was initially accomplished with the craft at the present loaded

weight of 5900 pounds.

Two additional masses totaling one thousand pounds were added along the centerline, one fore and one aft of the center of gravity. These masses constituted the control to attain a spectrum of pitch angles to be utilized in the simulation study. This is essentially equivalent to the method used when verifying the simulation results on the actual craft test runs. Ballast was shifted (in the form of warm bodies) to obtain the spectrum of pitch angles for verification of simulation results.

The bubble pressure in the XR-3 cannot be easily controlled, indeed it cannot be controlled at all. The plenum pressure can be reduced slightly by securing one or more lift supply engines, but a significant range of plenum pressures cannot be obtained. The lift fans operate at maximum speed at all times and the pressure obtained is approximately twenty-four pounds per square foot. Thus, only the middle pressure, twenty-four pounds per square foot could be verified. In the simulation, the bubble pressure was varied by changing the plenum supply fan speed. By this method, the actual power required to support the craft could easily be calculated.

On each simulation run, the speed of the craft was held constant and the thrust was allowed to vary to maintain the desired speed. The thrust was then utilized to calculate the thrust power in horsepower delivered. Additionally, for each run at a specific speed and bubble pressure, the pitch angle was varied by moving the masses along the longitudinal centerline and allowing the craft to attain a steady-state condition. The various data were then recorded for analysis and a next set of conditions was used to initiate a subsequent run, repeating the process.

In all cases, the simulation and actual craft operation was conducted above the transition speed, that is, the speed above which the craft acts as a Surface Effect Ship vice a displacement type vessel.

III. CALM WATER STUDIES

A. OBJECTIVES

The purpose of the calm water studies was to determine a data base to observe the general trends of the craft. Without sea state, the attainment of a steady-state pitch angle and operation could easily be obtained. This data is presented as Appendix A.

At each speed a family of curves was developed, each curve representing a new bubble pressure. Composites of all speeds are also presented, each taken at constant pitch angle and allowed to vary with bubble pressure. In each case, Total Power is the dependent variable.

B. SIMULATION PERFORMANCE

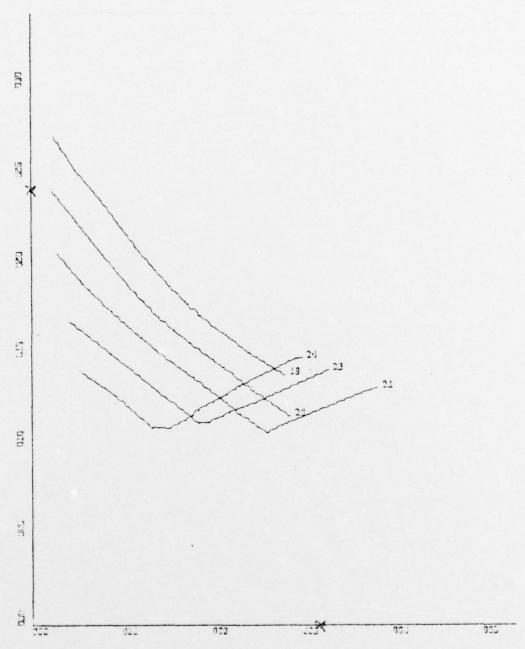
As can be seen in Figures 1 through 7, the total power (Thrust Power + Fan Power) reaches a minimum, or approaches a minimum, at each bubble pressure. The change in total power is relatively small at the lower cruising speed of fifteen knots, but is drastically reduced at the higher cruising speed of thirty knots.

In Figures 1 through 7, the ordinate is the Total Power expressed in actual horsepower delivered and the abcissa is Pitch Angle in degrees. At fifteen knots the curves tended

to overlap, therefore, for clarity, Figures 1 and 2 display data for this speed. In some cases, also, a minimum power could not be achieved. This is primarily at the higher pitch angles where water contact with the top of the plenum chamber occured, rendering these data inaccurate. At higher bubble pressures, the draft of the craft was quite small and relatively large pitch angles resulted in venting of the plenum to atmosphere. Again, these data were considered to be non-representative and were not included in the analysis.

The minimum power pitch angle at each speed is seen to move toward lower values as the bubble pressure is increased. This is felt to be a reasonable result in that the planing angle of the craft should be reached with a smaller angle as the draft decreases.

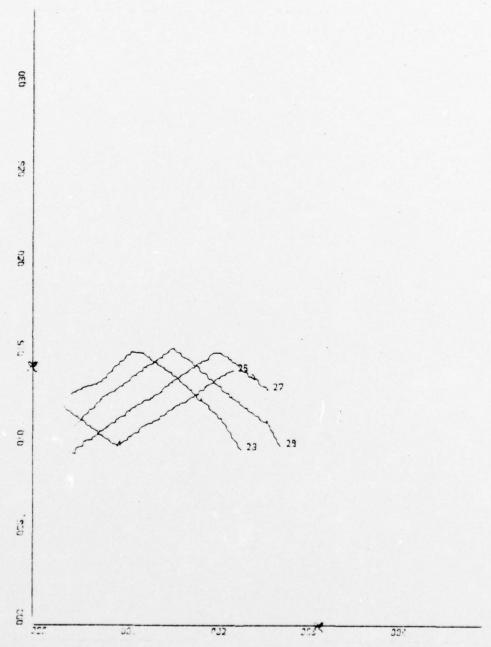
At the higher plenum chamber pressures an interesting and, at first glance, a somewhat unexpected phenomenon occurs. The slopes of the curves reverse and a local thrust condition appears to exist. accounted for by the shallow draft of the craft and the fact that so little of the sidewall is actually in the water (draft is about six inches at twenty-nine pounds per square foot). The craft, in this condition of operation, is approaching the behavior of an Air Cushion Vehicle. If the seals were large enough and stiff enough, eventually the craft would be completely above the water. flexible seal construction of the Captured Air Bubble craft, this condition is not possible. As the craft is pitched either way, the drag forces are decreased. In other words, the wetted area decreases on either side of an operating condition which corresponds to maximum wetted sidewall surface. This action is noted at all operating speeds.



Pigure 1 - TOTAL POWER VS PITCH ANGLE, 15.0 KNOTS

Curve Index: Plenum Pressure in PSF
X-Scale: 1.0 Deg/inch

Y-Scale: 0.5 HP/inch, Add: 20.0 HP to all values

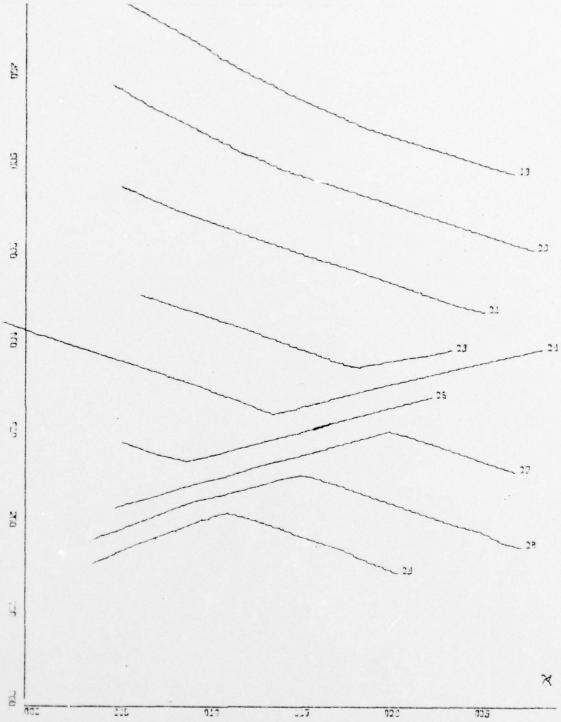


Pigure 2 - TOTAL POWER VS PITCH ANGLE, 15.0 KNOTS

Curve Index: Plenum Pressure in PSF

X-Scale: 1.0 Deg/inch

Y-Scale: 0.5 HP/inch, Add: 20.0 HP to all values

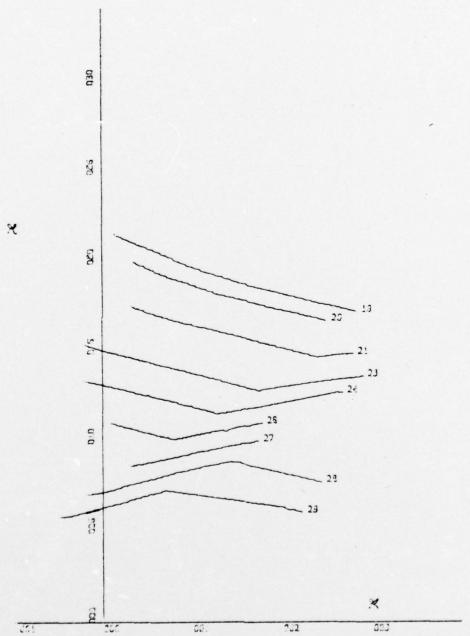


Pigure 3 - TOTAL POWER VS PITCH ANGLE, 18.0 KNOTS

Curve Index: Plenum Pressure in PSF

X-Scale: 0.5 Deg/inch

Y-Scale: 1.0 HP/inch, Add 23.0 HP to all values

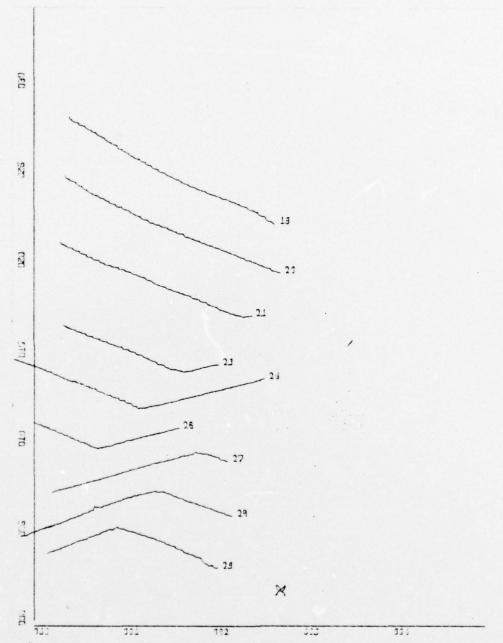


Pigure 4 - TOTAL POWER VS PITCH ANGLE, 22.0 KNOTS

Curve Index: Plenum Pressure in PSF

X-Scale: 1.0 Deg/inch

Y-Scale: 5.0 HP/inch, Add 25.0 HP to all values

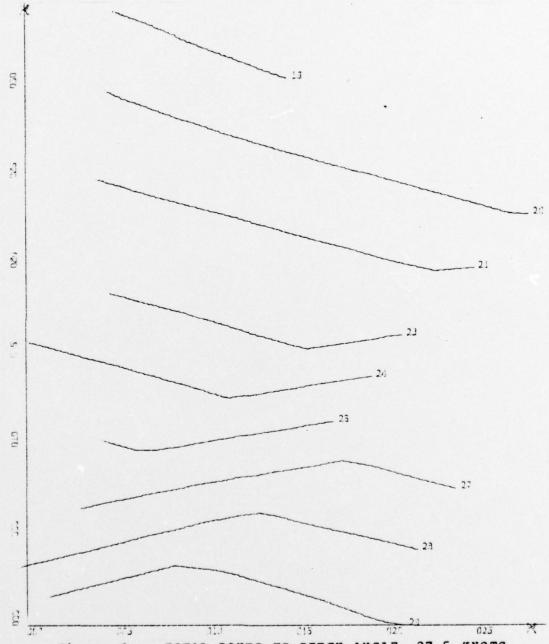


Pigure 5 - TOTAL POWER VS PITCH ANGLE, 25.0 KNOTS

Curve Index: Plenum Pressure in PSF

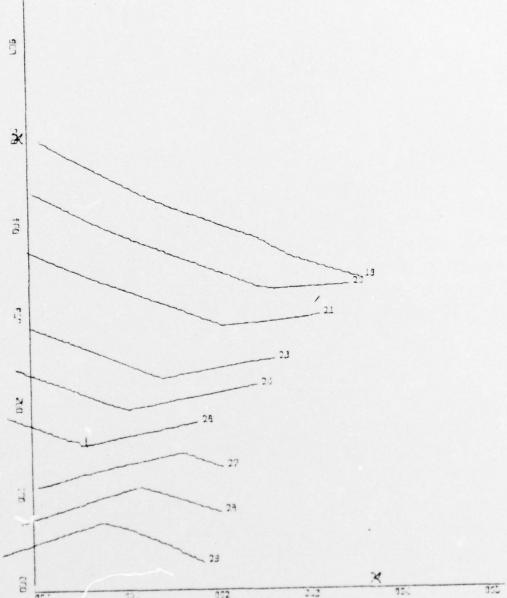
X-Scale: 1.0 Deg/inch

Y-Scale: 5.0 HP/inch, Add 25.0 HP to all values



Pigure 6 - TOTAL POWER VS PITCH ANGLE, 27.5 KNOTS
Curve Index: Plenum Pressure in PSF

Y-Scale: 0.5 Deg/inch Y-Scale: 5.0 HP/inch, Add 45.0 HP to all values



Pigure 7 - TOTAL POWER VS PITCH ANGLE, 30.0 KNOTS

Curve Index: Plenum Pressure in PSF

X-Scale: 1.0 Deg/inch

Y-Scale: 10.0 HP/inch, Add 50.0 HP to all values

Figures 8 through 28 display Total Power as a function of Plenum Pressure at the various cruising speeds. Each graph represents a slice at constant pitch angle obtained by linear interpolation of the existing data. This displays at each probable operating pitch angle the most efficient air plenum pressure. At the higher speeds, the most efficient operating plenum pressures are those in the higher range. The higher pressures result in lower hydrodynamic drag from a reduction in the sidewall-water interface contact. each curve is shown at a different pitch angle, planing action is observed to have a significant effect on total power above one degree pitch angle at speeds greater than twenty-two knots. At fifteen knots, the lowest total power is at approximately twenty-six pounds per square foot plenum pressure (compare Figures 2 and 12). The slight increase in total power at fifteen knots and large plenum pressures is caused by the fan power being approximately fifteen percent of the total power.

Again, minima can be seen to exist at each speed. The usefulness of this is explained in a later portion when a recommended operating profile is presented.

From the calm water studies, it can be seen that as the speed is increased to the higher cruising range, optimization is achieved by increasing the bubble pressure to the highest possible value, particularly with craft pitch angles above one degree (a very common operating point is one to two degrees).

At fifteen knots and below, the operating bubble pressure must be chosen very carefully at all pitch angles considered. Even at this low speed, proper choice of plenum pressure based on the steady-state pitch angle can result in a savings in power required of over six percent.

Once the spectrum of calm water runs over the range of pitch angles was completed, test runs were simulated for each speed at each plenum pressure to obtain the natural steady-state condition of the craft. This was accomplished by utilizing the moments for the X, Y and Z axes that been verified by previous studies of the XR-3 at the Naval Postgraduate School. The simulations were conducted under calm water conditions. At each speed, the lift supply fan speed was changed to yield the pressures utilized in the previous calm water simulations and the craft allowed to reach steady-state pitch angle and thrust. These results are shown graphically as Figure 29 for each bubble pressure. At the lower plenum pressures, the pitch angle does not vary significantly (0.4 degree) as the total power, and thus the speed of the craft, is increased. As the plenum pressure, however, is increased to the higher portion of the range, the steady-state pitch angle changes nearly 1.5 degrees as the total power is increased. Figure 29 also shows that the pitch angle and plenum pressure are essentially independent, especially at the lower range of pressures.

Figure 30 displays the same information at each speed. Note the considerable reduction in total power required to maintain a given speed as the plenum chamber pressure is increased from nineteen to twenty-nine pounds per square foot. From this graph, a one-third reduction in total power is realized along the thirty knot curve, where increasing plenum pressure allows total power to decrease from 85.44 to 56.52 horsepower. The power required to increase the pressure is only 1.61 horsepower.

C. EXPERIMENTAL VERIFICATION TESTS

Verification tests were conducted on the XR-3 craft

under calm water conditions. The total weight of the craft and ballast was 6895 pounds. The ballast was shifted along the longitudinal centerline to obtain a spectrum of pitch angles for comparison with the simulation results. The tests were conducted at fifteen, eighteen and twenty-two knots, constant speed. With the craft loaded this heavily, higher speeds could not be obtained. Only one air plenum pressure could be consistently obtained with the present configuration of the lift fan system. Figure 31 shows all three speeds at twenty-four pounds per square foot bubble pressure for the simulated and actual test runs for comparison. The same trends exist for both situations at each speed, thus producing the confidence in the simulation results to carry out the remainder of this study.

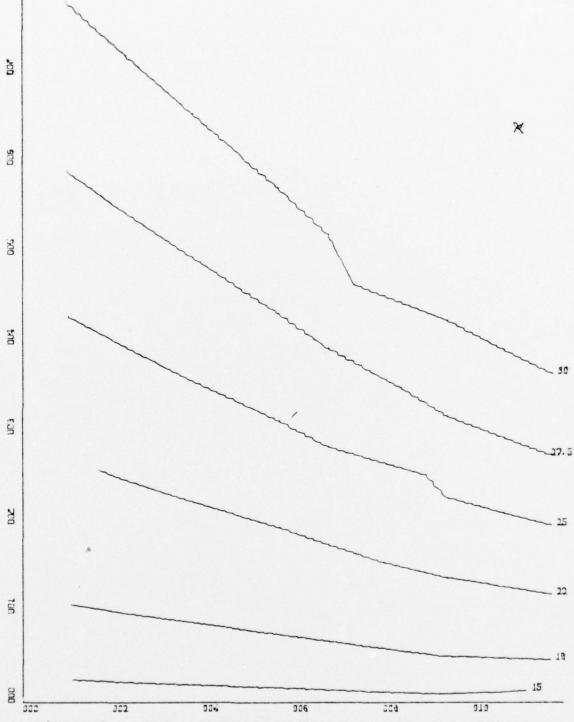


Figure 8 - TOTAL POWER VS PLENUM PRESSURE, 0.5 DEGREES
Curve Index: Speed in Knots

X-Scale: 2.0 PSF/inch, Add 18.0 PSF to all values Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values

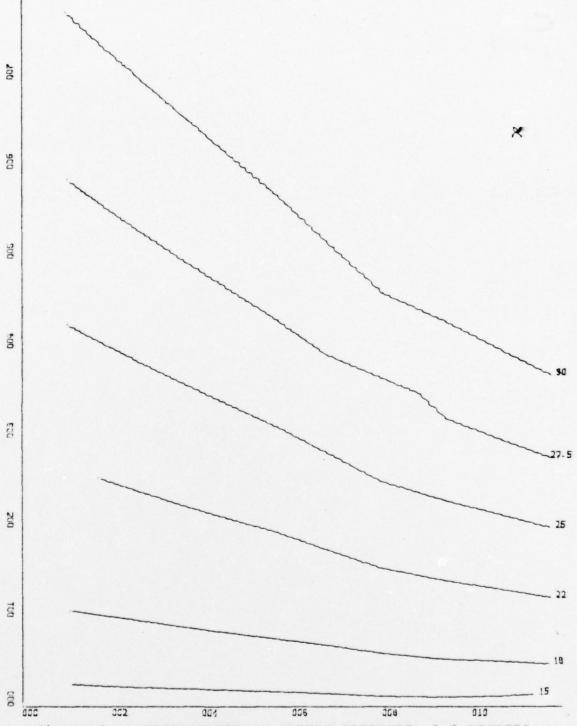


Figure 9 - TOTAL POWER VS PLENUM PRESSURE, 0.6 DEGREES

Curve Index: Speed in Knots

X-Scale: 2.0 PSF/inch, Add 18.0 PSF to all values Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values

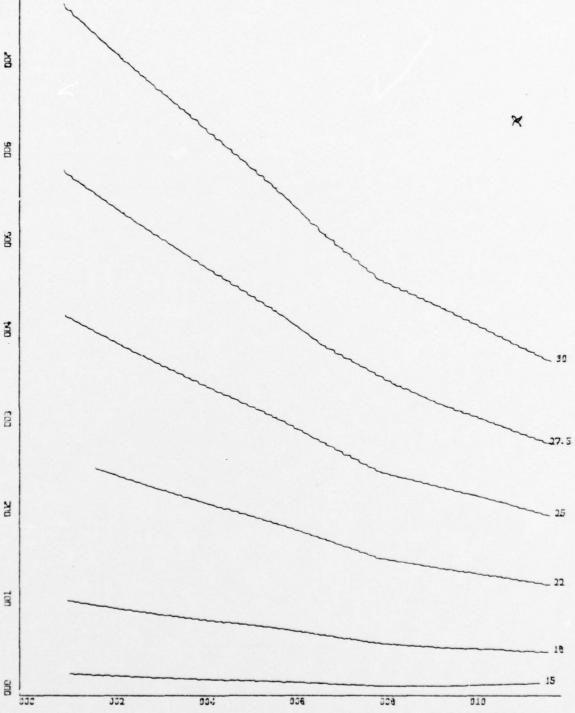


Figure 10 - TOTAL POWER VS PLENUM PRESSURE, 0.7 DEGREES

Curve Index: Speed in Knots

X-Scale: 2.0 PSF/inch, Add 18.0 PSF to all values
Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values

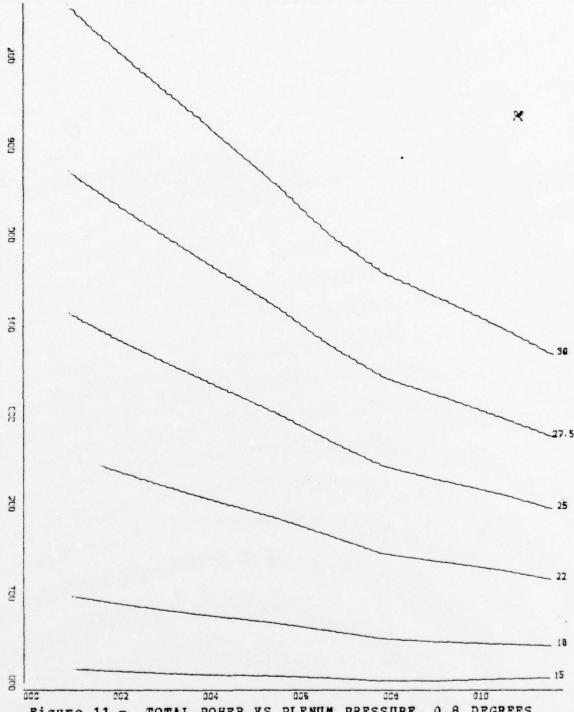
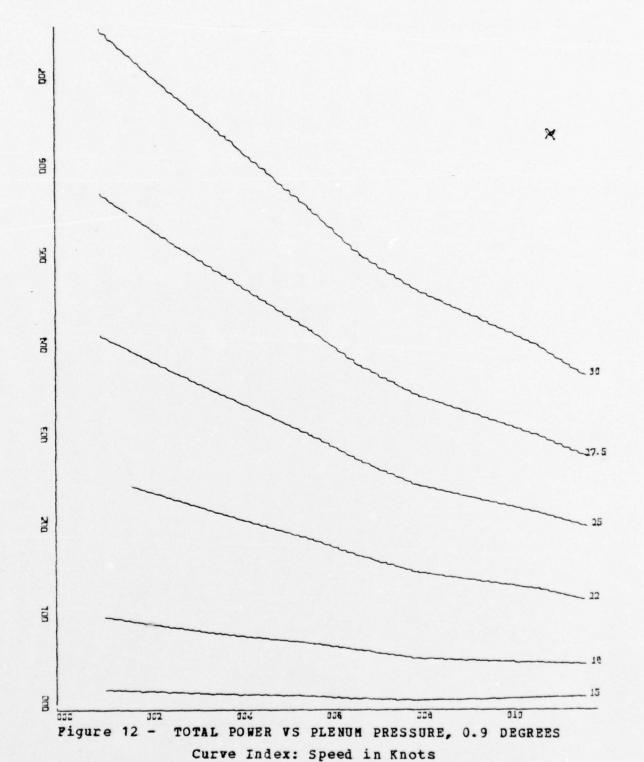


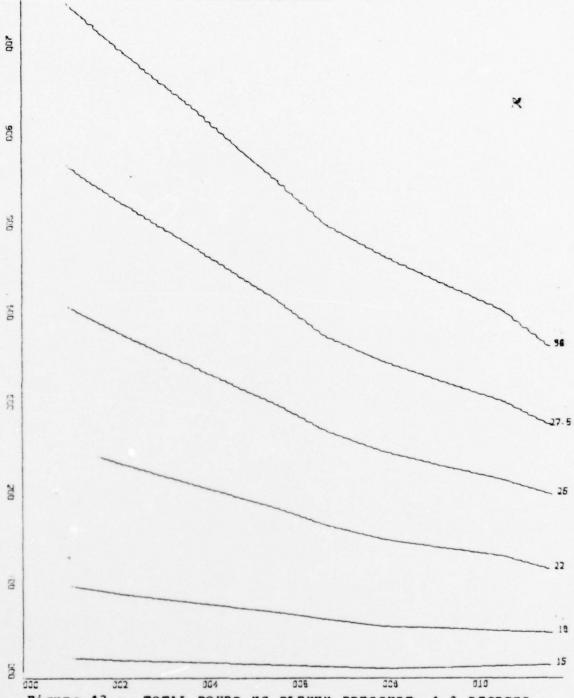
Figure 11 - TOTAL POWER VS PLENUM PRESSURE, 0.8 DEGREES

Curve Index: Speed in Knots

X-Scale: 2.0 PSF/inch, Add 18.0 PSF to all values Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values



X-Scale: 2.0 PSP/inch, Add 18.0 PSF to all values Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values



Pigure 13 - TOTAL POWER VS PLENUM PRESSURE, 1.0 DEGREES
Curve Index: Speed in Knots

X-Scale: 2.0 PSP/inch, Add 18.0 PSP to all values Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values

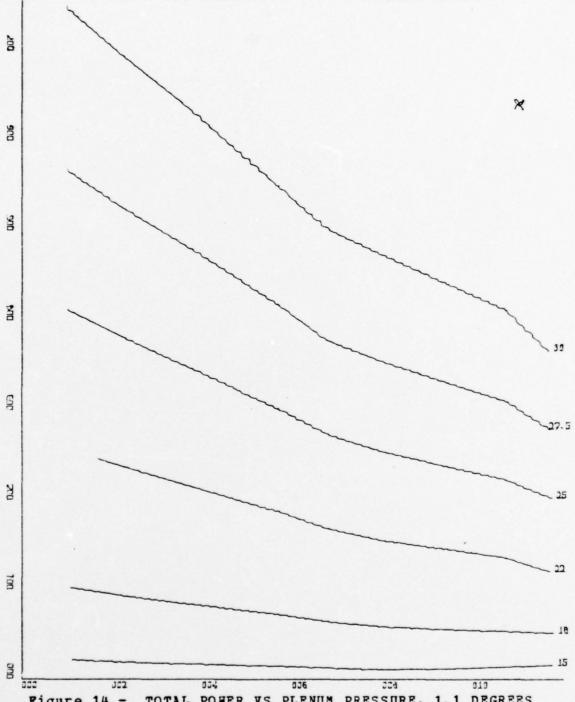


Figure 14 - TOTAL POWER VS PLENUM PRESSURE, 1.1 DEGREES

Curve Index: Speed in Knots

X-Scale: 2.0 PSF/inch, Add 18.0 PSF to all values Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values

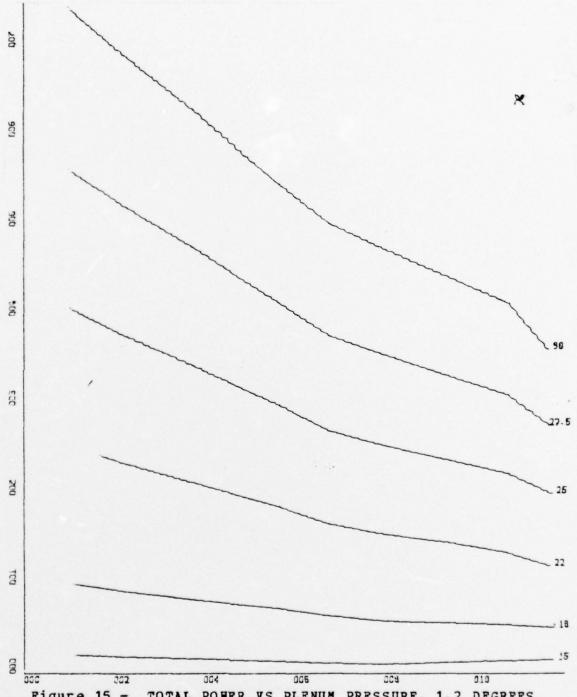


Figure 15 - TOTAL POWER VS PLENUM PRESSURE, 1.2 DEGREES

Curve Index: Speed in Knots

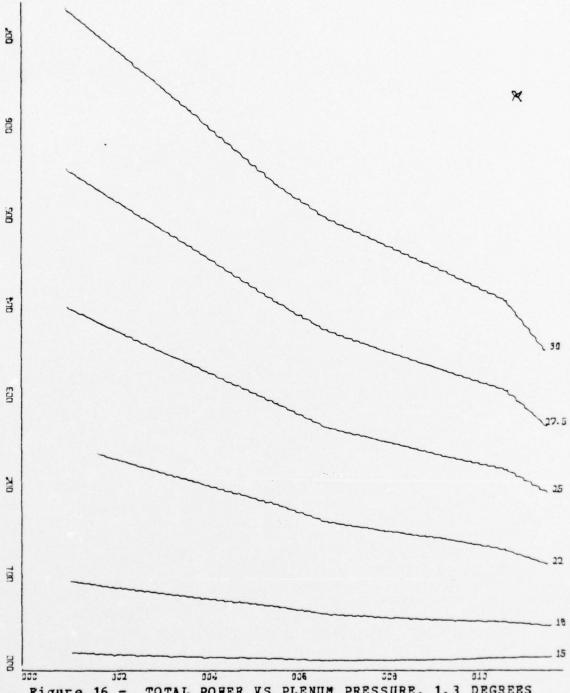


Figure 16 - TOTAL POWER VS PLENUM PRESSURE, 1.3 DEGREES
Curve Index: Speed in Knots

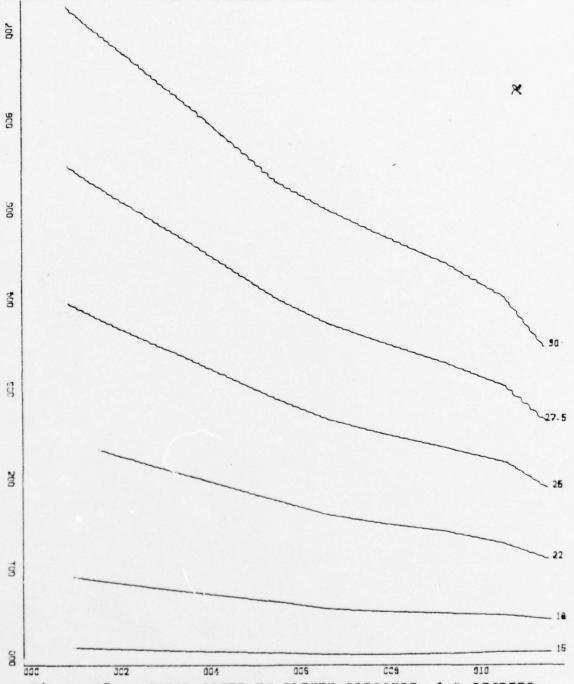


Figure 17 - TOTAL POWER VS PLENUM PRESSURE, 1.4 DEGREES
Curve Index: Speed in Knots

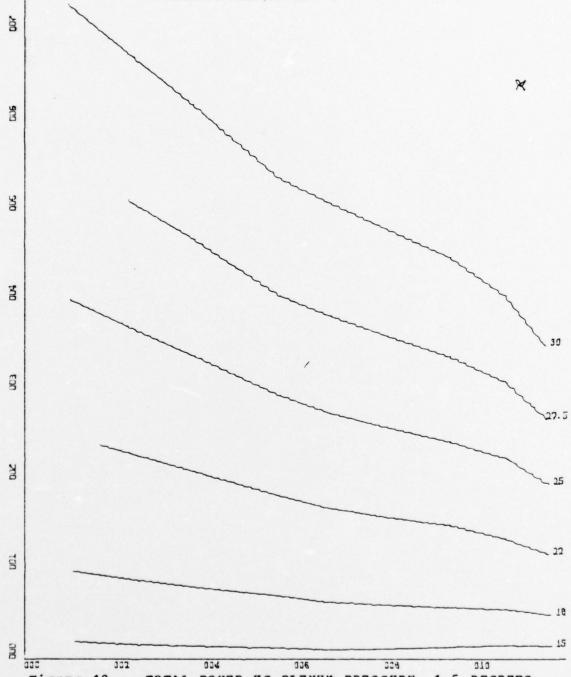


Figure 18 - TOTAL POWER VS PLENUM PRESSURE, 1.5 DEGREES

Curve Index: Speed in Knots

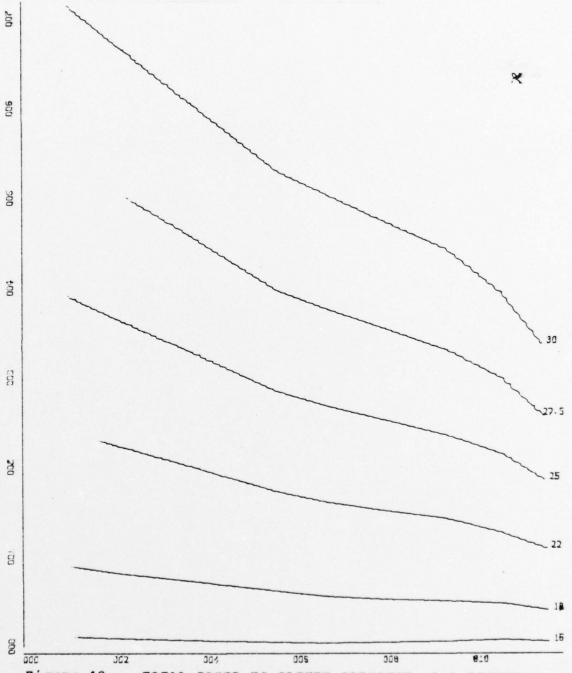


Figure 19 - TOTAL POWER VS PLENUM PRESSURE, 1.6 DEGREES

Curve Index: Speed in Knots

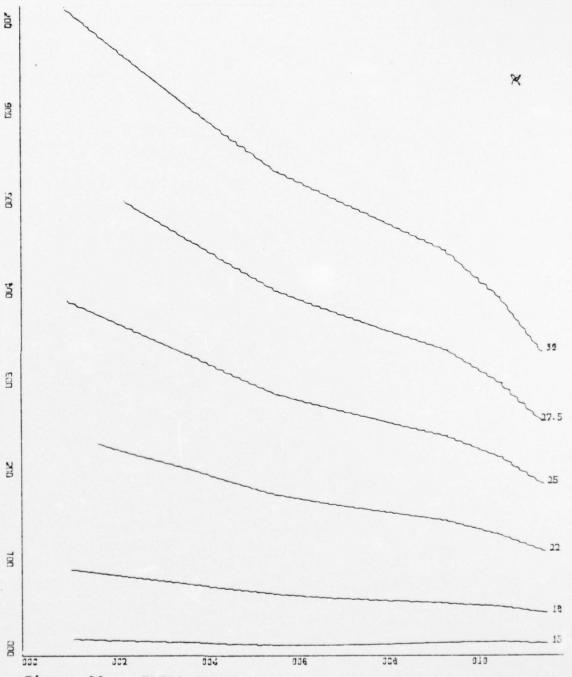


Figure 20 - TOTAL POWER VS PLENUM PRESSURE, 1.7 DEGREES

Curve Index: Speed in Knots

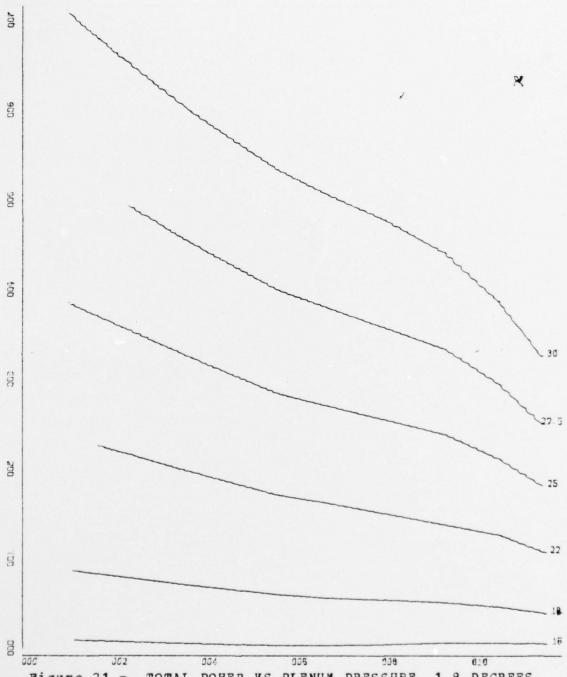


Figure 21 - TOTAL POWER VS PLENUM PRESSURE, 1.8 DEGREES
Curve Index: Speed in Knots

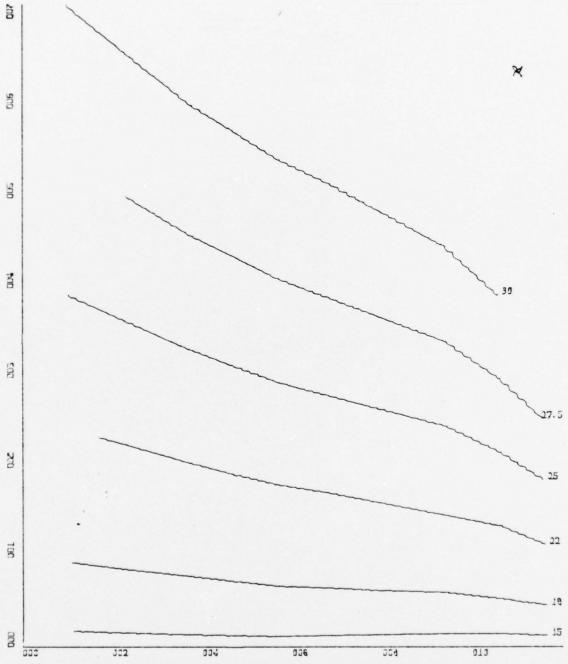


Figure 22 - TOTAL POWER VS PLENUM PRESSURE, 1.9 DEGREES
Curve Index: Speed in Knots

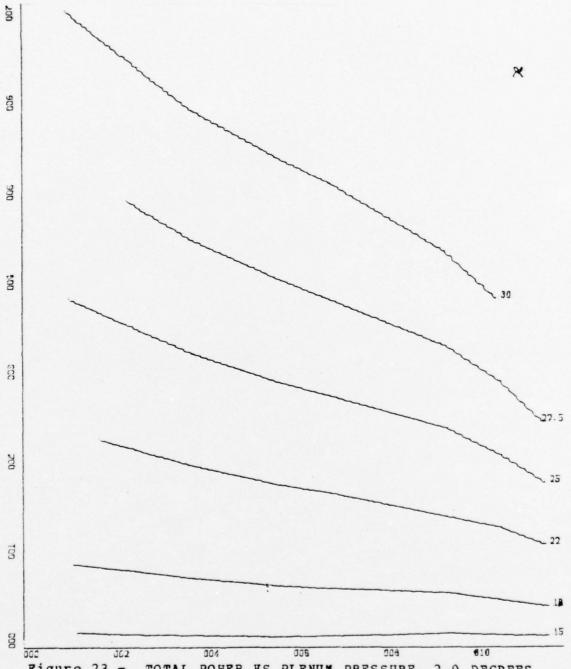


Figure 23 - TOTAL POWER VS PLENUM PRESSURE, 2.0 DEGREES
Curve Index: Speed in Knots

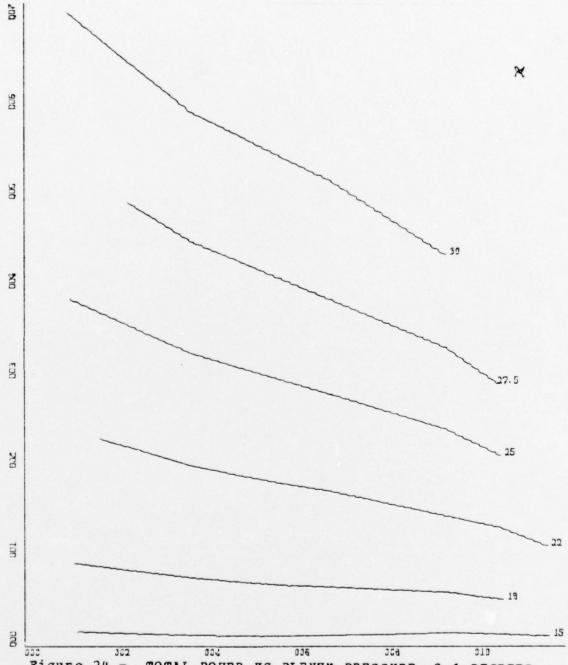


Figure 24 - TOTAL POWER VS PLENUM PRESSURE, 2.1 DEGREES

Curve Index: Speed in Knots

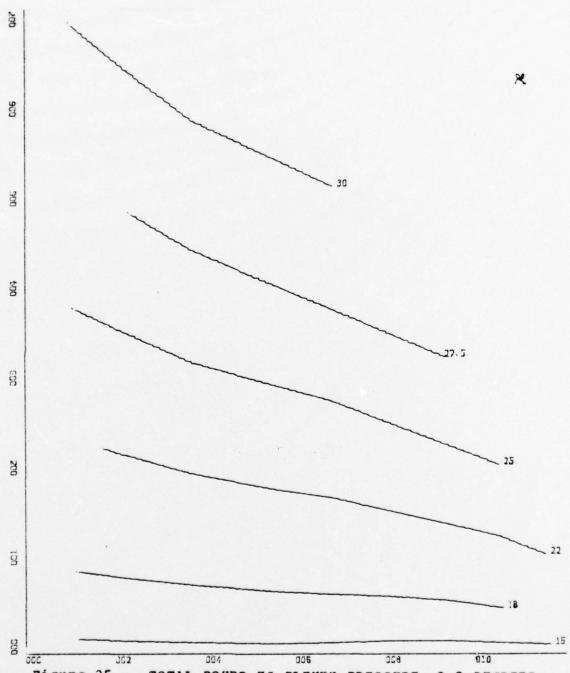
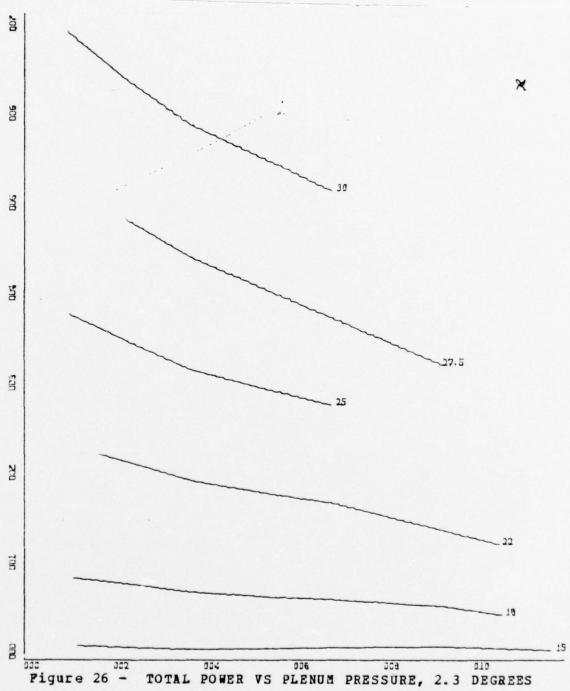


Figure 25 - TOTAL POWER VS PLENUM PRESSURE, 2.2 DEGREES

Curve Index: Speed in Knots



Curve Index: Speed in Knots

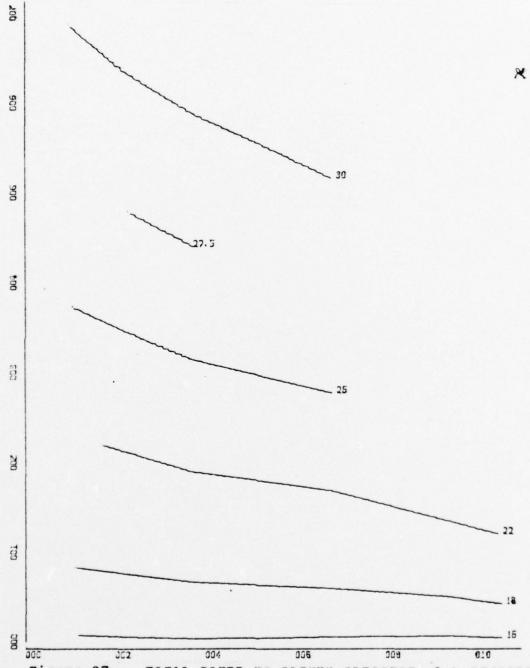


Figure 27 - TOTAL POWER VS PLENUM PRESSURE, 2.4 DEGREES
Curve Index: Speed in Knots

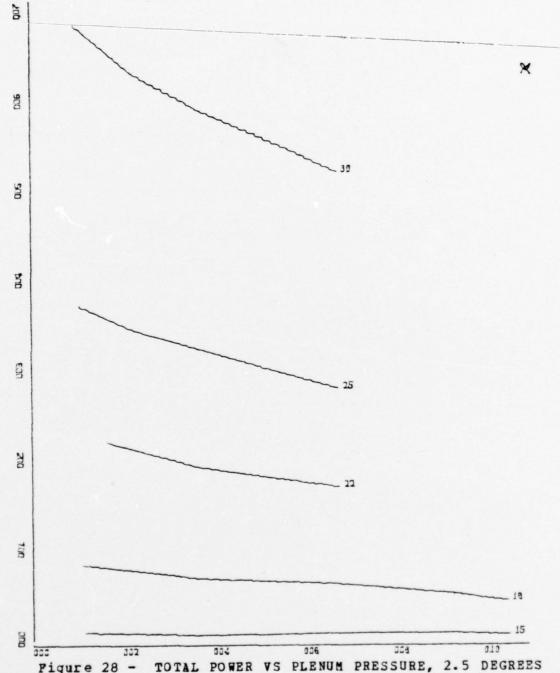


Figure 28 - TOTAL POWER VS PLENUM PRESSURE, 2.5 DEGREES

Curve Index: Speed in Knots

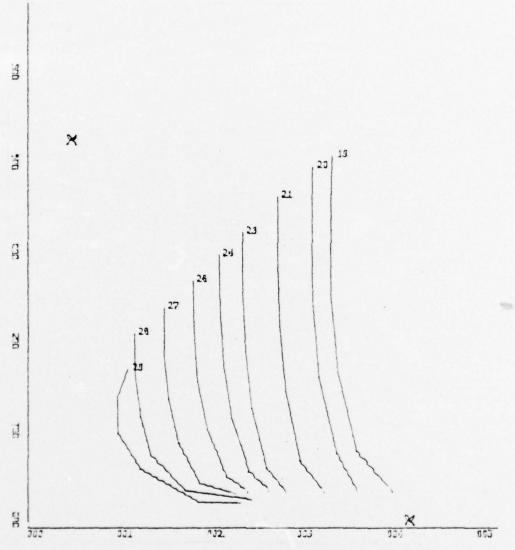


Figure 29 - TOT PWR VS PITCH ANGLE, NATURAL RESPONSE

Curve Index: Plenum Pressure in PSP

X-Scale: 1.0 Deg/inch

Y-Scale: 10.0 HP/inch, Add 30.0 HP to all values



Figure 30 - TOT PWR VS PITCH ANGLE, NATURAL RESPONSE Curve Index: Speed in Knots

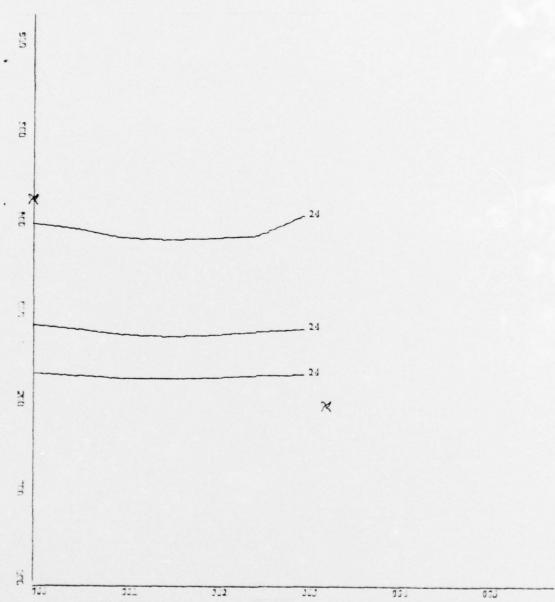


Figure 31 - TOT PWR VS PITCH ANG, ACTUAL CRAFT

Curve Index: Speed in Knots

X-Scale: 1.0 Deg/inch Y-Scale: 10.0 HP/inch

IV. SEA STATE STUDIES

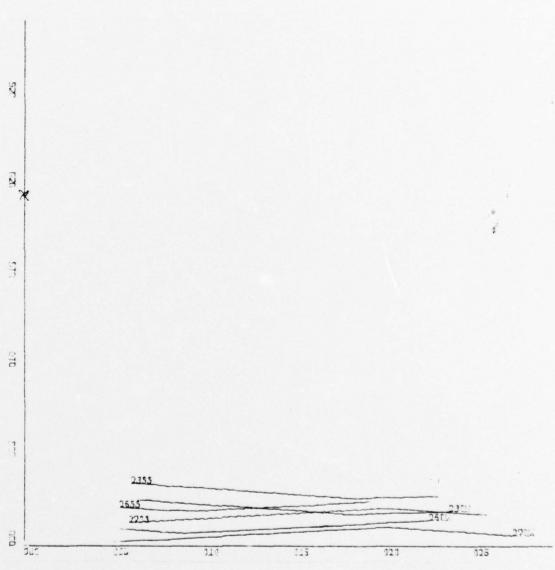
A. OBJECTIVES

Once the calm water studies were completed, sea state was introduced into the Loads and Motions Program for the XR-3 to continue the study. The sea state simulation studies were conducted at two speeds, one at 27.5 knots, the other at 18.0 knots. Because of a nearly sixty to one computation time to real simulation time ratio, an exhaustive study was prohibitive. General trends with representative sea state introduced was desirable to be compared with the calm water simulation runs.

B. SIMULATION PERFORMANCE

The introduction of sea state was accomplished by using a single wave component with frequency 0.7662 radians per second and height of one foot peak-to-peak. A single component sea state such as this is termed a regular sea, which was chosen to obtain reasonable computational times. Regular seas were also selected to allow somewhat easier data reduction. The data was smoothed to obtain an average value of each parameter and this average value was utilized as the steady-state value.

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Pigure 32 - TOTAL PWR VS PITCH ANGLE, 18 KNOTS, SEA STATE

Curve Index: Plenum Pressure in PSF

X-Scale: 0.5 Deg/inch

Y-Scale: 5.0 HP/inch, Add 25.0 HP to all values

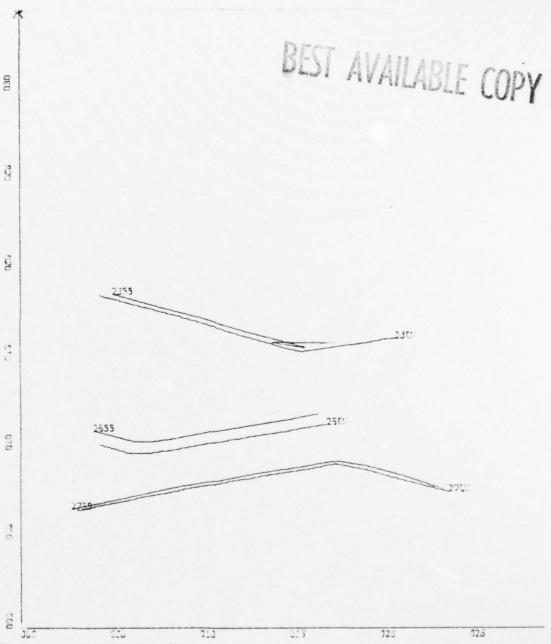


Figure 33 - TOTAL PWR VS PITCH ANGLE, 27.5 KNOTS, SEA STATE
Curve Index: Plenum Pressure in PSF

X-Scale: 0.5 Deg/inch
Y-Scale: 5.0 HP/inch, Add 45.0 HP to all values

To obtain a spectrum of pitch angles, the same procedure of shifting masses along the longitudinal centerline was used as in the calm water study. The total weight of the craft in the simulation remained unchanged.

Although not as detailed, Figures 32 and 33 display the same general trends as found in the calm water studies. As plenum pressure is increased, the resultant total power required to maintain the desired speed decreases. The total power at each bubble pressure is seen to increase over the same calm water condition. The plenum pressures for the sea state data are shown on the left side of each curve and marked with SS and the calm water plenum pressures are at the right side designated with CW. In all cases, the total power is slightly greater (4 to 6 percent) for the sea state tests. This is an expected and reasonable phenomenon. The wetted sidewall surface is now irregular causing an increase in average thrust power required to maintain the specified speed. The average fan power is also seen to increase in an attempt to maintain the bubble pressure constant.

The operational pitch angles are much more restricted than in the calm water simulations. With one foot waves, venting of the plenum or water contact with the plenum top occurs much more readily at the plenum pressure extremes. Bubble Pressures below twenty-one pounds per square foot allowed frequent contact with the air plenum top surface and at the higher bubble pressure (above twenty-seven pounds per square foot) excessive venting occurred at Pitch Angles above 2.5 degrees and below 0.3 degrees.

For the purposes of this study, three plenum pressures were utilized: 23, 26 and 27 pounds per square foot. This choice of pressures allowed representative trends to be observed without plenum chamber water contact or excessive

plenum venting. A comparison is made of the calm water and sea state simulations. Figures 32 and 33 display the slightly higher total power necessary to operate the craft in a sea state condition, the increase in power being approximately six percent over the total range of pitch angles used. From this comparison, it is concluded that the craft, in sea state conditions, operates in much the same manner as in calm water, therefore, all further analysis is conducted for calm water conditions. All conclusions and recommendations will be equally applicable to sea state operation.

V. RESULTS

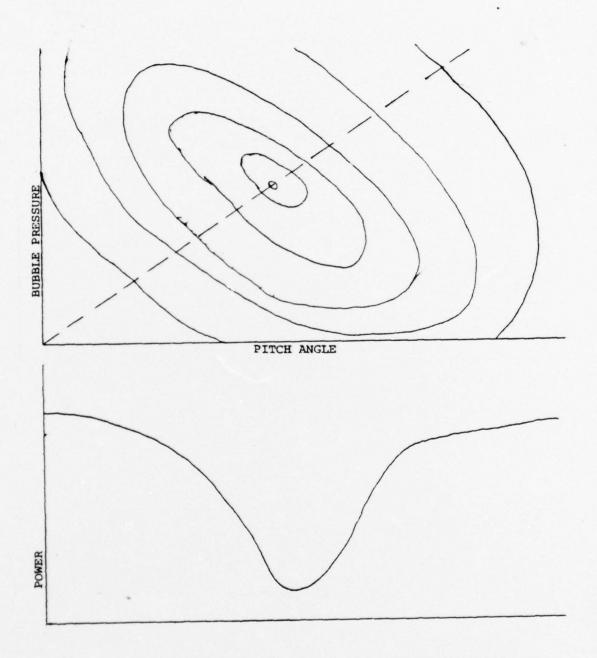
Most data has been presented in the sections devoted to each specific type of simulation condition conducted. To test for a global minimum, air plenum pressure was plotted against pitch angle with total power held constant. If a global minimum is to exist as a function of the two control variables, one would expect somewhat concentric circles or concentric contours at each constant total power point.

The sketches presented as Figure 34 represent the global minimum expected at each speed. The contours of sketch A represent lines of constant total power on a plot of Plenum Pressure versus Pitch Angle. As the optimum operating point is approached (movement toward the central contour), the range of plenum chamber pressures and pitch angles become more restricted. Sketch B of Figure 34 is a view perpendicular to the dashed line shown on sketch A. It represents the profile of total power based on chosen values of plenum pressure and pitch angle along that dashed line and displays the actual minimum power point.

From the data produced in the simulations, only fifteen knots speed can be analyzed in this fashion. At all other speeds, the bubble pressures chosen for analysis were too far apart and did not produce a sufficient number of points at each constant power level to produce a graph. Figure 35 represents the graph of Air Plenum Pressure versus Pitch Angle for fifteen knots. Curves A through I indicate plenum pressures of 19, 20, 21, 23, 24, 26, 27, 28 and 29 respectively. As shown by the flatness of these curves, bubble pressure is nearly independent of pitch angle over

the operational range of pitch angles chosen for this study. This is, of course, a desirable feature as these are the two control variables. Also shown are contours of constant total power. These are in the range of 21.2 to 21.8 horsepower. The solid portion of the contours represent the actual data while the dotted portion is extrapolated. data for the contours was obtained by linear interpolation of the calm water data at constant total power. From Figures 1 and 2 we would expect two different pitch angles to yield the same total power at several of the plenum pressures, which produces the contours shown. For fifteen knots, Figure 35 displays a global minimum at 1.7 degrees pitch angle and 24.1 pounds per square foot plenum pressure, as determined from Figures 1 and 2. The minimum power point, and thus the point of greatest efficiency, is determined graphically to be approximately 20.9 horsepower.

Based on the similarity of results between the calm water simulations and the simulations after the introduction of sea state, a similiar global minimum should be obtained under sea state conditions. It is expected that the global minimum would occur at a slightly higher total power than that of similar operation in calm water, however it would occur at essentially the same pitch angle and plenum pressure.



Pigure 34 - SKETCHES OF THE GLOBAL MINIMUM CONDITION

A - Sketch of Bubble Pressure vs Pitch Angle

B - Section Along Dashed Line in (A)

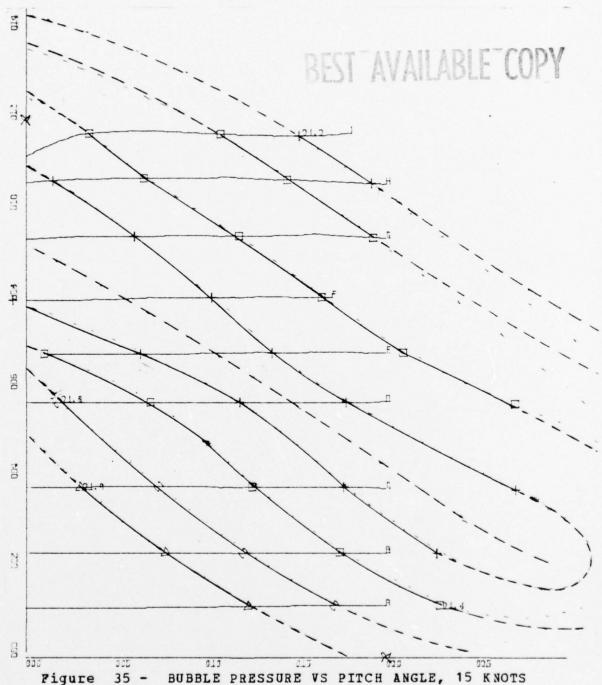


Figure 35 - BUBBLE PRESSURE VS PITCH ANGLE, 15 KNOTS Contours: Total Power in HP

X-Scale: 0.5 Deg/inch, Add 0.5 Deg to all values Y-Scale: 2.0 PSF/inch, Add 18.0 PSF to all values

VI. CONCLUSIONS

A. SUMMATION

At each speed an optimum operating point exists where efficiency can be maximized. This increase in efficiency (up to forty percent is possible) can result in a considerable savings in operating costs, or, possibly more importantly in a military application, extend the operating range of the craft. For example, at thirty knots and a craft pitch angle of 1.5 degrees, simply increasing the plenum pressure from nineteen to twenty-nine pounds per square foot results in the indicated savings of forty percent in total power required for operation of the XR-3. This is shown in Figure 36. The high speed capabilities of this type of craft have been previously demonstrated, and it is concluded from this study that optimization can be achieved over the full range of cruising speeds, but most significantly at the higher speeds.

In general, the power required to support the craft is relatively independent of the forward speed thrust power at all speeds. The data of Appendix A demonstrates this very well. Under the column heading FAN PWR, the actual power required to supply the necessary lift pressure is seen to be nearly constant over the entire speed range at each bubble pressure. Note also that the fan power does not change as the craft speed or pitch angle changes, only when the plenum pressure is altered. Therefore, it is prudent to increase the lift fan power supplying the plenum pressure at higher

speeds to effect a decrease in drag forces. At the lower cruising speeds, lift fan power becomes a significant factor (approximately fifteen percent) of total power, and thus the bubble pressure must be chosen carefully based on the pitch angle to obtain optimal operation and power efficiency. The pitch angle and bubble pressure must be utilized in harmony to achieve this optimization.

The pitch angle of the craft is also seen to be a significant factor in power optimization. From Figure 36, for example, operation of the craft at twenty-four pounds per square foot pressure and 1.1 degrees pitch angle requires only 70.1 horsepower. At all other pitch angles, the required power increases. A savings of six percent, under these conditions, can be realized if the optimal pitch angle is utilized.

B. METHOD OF CONTROL

Operator control of both pitch angle and plenum pressure is certainly a realizable method of obtaining optimal operation of the craft. It would, however, require a complete set of information on every possible combination of operational attitudes of the craft. Although it could be stored as a set of operational profiles in a digital computer to be recalled at the will of the operator, this is prohibitive because of the computational time required to obtain such a wide range of data. Additionally, no two craft operate exactly the same, each having its own peculiarities. It is concievable that a separate set of profiles would have to be produced for each ship in the class.

Automatic control of both pitch angle and air plenum

pressure is also a possibility. The physical method of controlling these two variables will not be considered here, but the demonstrated optimization could be achieved with a minimum power seeking control system. It is envisioned to be a system with input parameters, in addition to the attitude of the craft, of thrust power and fan power. At a given total power level, the pitch angle of the craft could be perturbed slightly by the control system. disturbance resulted in a reduction in total power, the perturbation would continue until a further disturbance resulted in a power increase. A similar set of perturbations would then be introduced into the plenum pressure system and, again, power minimization sought. Once the minimum power point, and thus the optimal operating point, is attained, this two parameter control system would maintain optimization throughout craft operation.

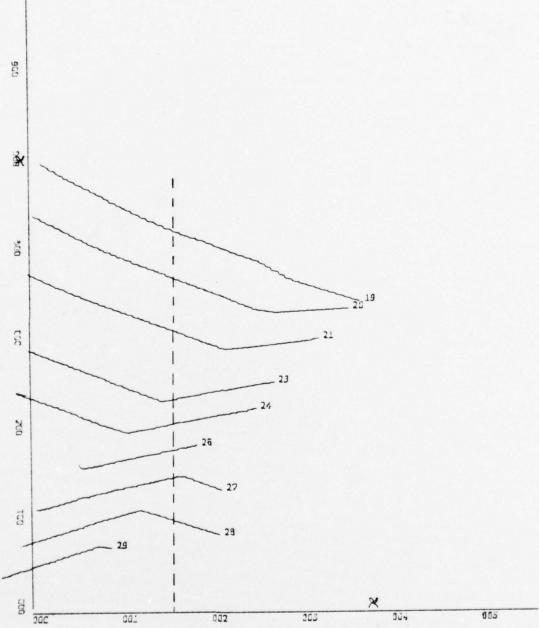


Figure 36 - TOTAL POWER VS PITCH ANGLE, 30 KNOTS

Curve Index: Plenum Pressure in PSF

X-Scale: 1.0 Deg/inch

Y-Scale: 10.0 HP/inch, Add 50.0 HP to all values

VII. OPERATIONAL CONSIDERATIONS

At the present time, pitch control is not incorporated into any of the Captured Air Bubble Test Vehicles. This includes both the three ton and the one hundred ton models. Until such time as pitch control is made available, it is recommended that the plenum pressure be adjusted once the steady-state pitch angle is achieved at the desired cruising speed.

Figure 37 is presented as the Optimum Operating Profile. Again, Total Power is plotted aginst Pitch Angle and two speeds, eighteen and twenty-seven knots are shown for comparison. Only calm water data are presented since they are representative of the sea state conditions as well. Several cases are considered and the use of this information is presented below:

1. Consider the case of non-optimal initial conditions. The craft is operating at 27.5 knots, 26 PSF air plenum pressure. The natural pitch angle of the craft is 2.25 degrees (Point A). To optimize under these comditions, the pitch angle should be changed to 0.7 requiring 3.75 horsepower less than the original condition. This is shown as Point B. To optimize still further, the air plenum pressure should increased to 29 PSF (Point C) reducing the power required to maintain 27.5 knots by an additional 6.9 horsepower. If optimization is continued, the pitch angle should be altered to arrive at Point D, resulting in a total reduction in required power of horsepower, or twenty-five percent.

- 2. While operating at Point D, it is desired to change speed to eighteen knots. Thrust power is reduced to allow the craft speed to decrease to eighteen knots while maintaining 2.0 degrees pitch angle, and the craft is at an optimal power level (Point E). Note, however, that the craft could also operate at Point F, with a pitch angle of 0.4 degrees and still remain at the optimal power level. This would be operator choice, and might be considered for reasons of crew preference or equipment operation.
- 3. Assuming pitch control is not available (as is the present situation), at 18.0 knots and 29 PSF plenum pressure, the natural pitch angle is 1.9 degrees, or essentially the optimal pitch angle (Point E). If the speed were increased to 27.5 knots with the plenum pressure unchanged, the craft would naturally assume a 1.0 degree attitude (Point G) which is very near the maximum power level for this plenum pressure. It has been noted in both simulation and actual operation that the craft does not necessarily assume the optimal attitude. In actual operation, for the given power level, the craft could have just as easily The perturbations during the settled at Point C. transition control this phenomonon and it is mentioned purely because it does exist in craft operation.

Two specific speeds were utilized for these examples, but any combination of speed and/or air plenum pressure changes can be studied in similar fashion by use of Figures 1 through 7 in the same manner as Figure 37. Since pitch control is not available, one must use whatever pitch angle is assumed by the craft and optimize operation by altering the plenum chamber pressure accordingly.

If pitch angle and plenum chamber pressure control were

both available, the constant power contours of Figure 38 could be used to determine the point of optimal operation. Figure 38 is for fifteen knots, but, as explained earlier, similar results exist at all speeds. At fifteen knots, the operating profile (Figure 38) would require choices of bubble pressure and pitch angle to reach the center contour. This could be accomplished either by operator (manual) control or an automatic controller. For this speed, pitch angle would be adjusted to 1.7 degrees and the air plenum pressure to 24.1 PSF to operate the craft at minimum total power, 20.9 horsepower. This is the optimal point of operation at fifteen knots. The broadness of this minimum could not be determined from the existing data, but the trend indicates it is relatively small. Based on the similarity of calm water and sea state studies, this minima is not expected to broaden nor change significantly.

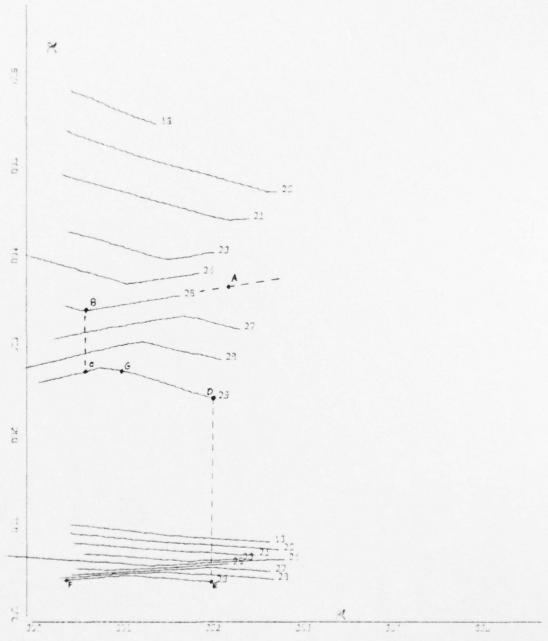


Figure 37 - TOT PWR VS PTCH ANG, 18 AND 27 KNOTS

Curve Index: Plenum Pressure in PSF

X-Scale: 1.0 Deg/inch

Y-Scale: 10.0 HP/inch, Add 20.0 HP to all values

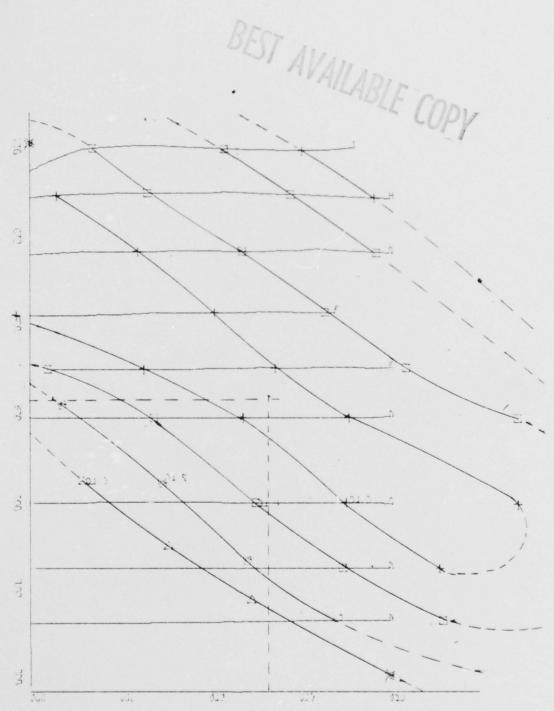


Figure 38 - RECOMMENDED OPERATING PROFILE, 15 KNOTS
Contours: Constant Total Power in Horsepower
X-Scale: 0.5 Deg/inch, Add 0.5 Deg to all values
Y-Scale: 2.0 PSF/inch, Add 18.0 PSF to all values

VIII. RECOMMENDATIONS

As demonstrated, control of both plenum pressure and pitch angle are necessary to achieve power optimization. Air plenum pressure control is incorporated into the larger, one hundred ton models by use of variable speed lift fans. Pitch angle control, although not presently available, could be achieved by a ballast shifting system, possibly using water and/or fuel tanks distributed along the sidewall length. An alternate method would be to use additional controllable surfaces placed below the waterline to effect pitch control.

A major consideration which is beyond the scope of this study is the determination of whether pitch control introduced should be automatic or simply a set of recommended pitch angles controlled by an averaging system by the operator. This is left as a possible future thesis subject.

Optimization of power can be achieved only through judicious choice of both pitch angle and plenum chamber pressure. The method of control may be either manual or automatic, but the results display a considerable savings if the Captured Air Bubble Surface Effect Ship is operated toward this optimal goal.

APPENDIX A

SIMULATION DATA LISTING

Appendix A is presented as the calm water data produced by the Loads and Motions Program for the XR-3 Surface Effects Ship. This is done to allow the reader to obtain specific parameter values used to present the graphical information included in the body of the text. It is also presented because it represents considerable computation time (nearly one hundred hours of computer time). The data can therefore be utilized in subsequent analysis work if desired.

The column headings are presented at the top of each page. Pitch angle (THETA) is in degrees, THRUST is expressed in pounds, BUBBLE PRESSURE in pounds per square foot, and FAN POWER, THRUST POWER and TOTAL POWER in horsepower.

The graph titles remain in the listing to allow ease in separation of the data as are the graphical set-up cards separating each bubble pressure at each speed. The four rightmost digits are for rapid identification based on speed (the first two digits) and bubble pressure (the last two).

	TOT PWR	19.9917	22. 3744 22. 3744 22. 1797 22. 1797 21. 89846 21. 80929 21. 5573 21. 5188	22. 23. 22. 25. 22. 22. 22. 22. 22. 22. 22. 22	222 222 221 221 221 221 221 221 231 333 231 231
0 KNDTS	THST PWR	17.4917	200.000 00 00 00 00 00 00 00 00 00 00 00	200. 200. 200. 200. 200. 200. 200. 200.	199. 199. 199. 199. 199. 199. 199. 199.
•	BUB PRES	25.0000	19.10879 19.10879 19.11620 19.112622 19.113622 19.115584 19.115565 19.116468	20202020202020202020202020202020202020	221.881223 221.8812623 221.88203 221.881001 221.881001 221.881001 221.881001 221.881001
VS PITCH	FAN PWR	2.5000	1.7268 1.7266 1.7262 1.7262 1.7263 1.72664	11.091234 11.0912301 11.0912301 11.0912301 11.091231 11.091231	22.1.28873 22.1.28873 22.1.28874 22.1.28874 22.1.2888
OTAL POWER BBLE PRESS	THRUST	380.0000	4452.2134 4448.5698 4444.3671 442.0918 442.0918 4436.1038 4436.1038 4437.94822 4597.94822 4597.94831	445 441.838804 437.88738 437.80906 432.2212 428.7493 428.7493 425.6946 421.3799	433.3992 426.2913 423.1382 421.5957 421.5957 418.7776 413.7776 411.7715
FIGURE 1, T	THETA	23.200	84080400000000000000000000000000000000	1512 0.012 0.02255 0.080159 1.009887 1.058836 1.05886 1.0588	76 11 1980 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1

TOT PWR	21.3089	1.708 1.566 1.268	1126	21.1837 21.2138 21.2431 21.4014 21.4234	21.2792 21.2792 21.1029 21.0974	1.142	11.33.84
THST PWR	19.1835	9.316	88888 7773 7230 7300 700 700 700 700 700 700 700 700	18.7923 18.8216 18.8527 19.0039	18.8664 18.7218 18.5457 18.5427	88888 8000 8000 8000 8000 8000 8000 80	8.926 8.926 8.937
BUB PRES	21:7112	707	7759	23.7151 23.7231 23.6646 23.6445	24.7927 24.7847 24.7866 24.8069 24.8149	444	4447637
FAN PWR	2.1254	3922	39991	2.3914 2.3922 2.3904 2.3975 2.3999	22.55	COCOCO COCOCOCO COCOCOCO COCOCOCO COCOCOCO COCOCOCO COCOCOCOCO COCOCOCOCOCO COCOCOCOCOCO COCOCOCOCOCOCO COCOCOCOCOCOCOCO CO	550000 500000 500000
THRUST	416.7532	19.632	07 048 07 023 07 405 07 176	408.2554 408.8909 409.5662 412.8506 413.2769	409.8633 406.7234 402.8975 402.8313 403.3887	003 . 842 005 . 005 005 . 005 005 . 005 005 . 005	06.955 08.025 11.176
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THS	88888888888888888		
BUB PRES	22222222222222222222222222222222222222	2000 2000	8 .515 999999 8 .515 9999999 9 .5965
FAN PWR		0.000000000000000000000000000000000000	195 141 141 141 141 141 141 141 141 141 14
THRUST	3997 39987 39987 39987 4000 4000 4000 4002 400	BBBBBBBBBBBBBBBBBBBBBBBBBBBBBBBBBBBBBB	86.870 888.162 888.513 888.772 889.589
THETA	44100000000000000000000000000000000000	22222222222222222222222222222222222222	74440 7440 7740 77

TOT PWR	211.3820 221.4526 221.4526 221.52454 221.25398 221.25398 221.25398 231.125898 20.3846
THST PWR	17.9743 18.09463 18.00371 18.1075 18.114442 18.114442 17.88716 17.88716 17.76800
BUB PRES	29.71746 29.71746 29.77246 29.77510 29.75510 29.6560 29.6560 29.6560 29.6560
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THST PWR	27 27 27 27 27 27 27 26 26 26 26 26 26 26 26 26 26 26 26 26	255 4 3 4 4 4 4 5 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6	25.12065 25.1730 25.1730 25.1586 25.0947 25.0018
BUB PRES	20.20.20.20.20.20.20.20.20.20.20.20.20.2	21.880881 22.1.380881 22.1.381001 22.1.381001 22.1.381001 22.1.381001 23.1.381001 23.1.381001 23.1.381001 24.1.381001 25.1.381001 25.1.381001 26.1.381001 27.1.381	23.7131 23.7131 23.7090 23.7090 23.7090 23.7090
FAN PWR	11.09.09.09.09.09.09.09.09.09.09.09.09.09.	2007-2007-2007-2007-2007-2007-2007-2007	22.39917. 22.39922. 23.39222. 23.39222. 23.392. 23.3922. 23.302. 23.302. 23.302. 23.302. 23.302. 23.302. 23.302
THRUST	493.3721 492.6093 492.6093 690.4866 487.3506 485.21506 485.21506 476.7295 476.4756 476.4756	44448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 448833 4488	456.3335 456.0342 455.7266 455.4661 454.3101 453.4517 452.6270
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THS	39		999999
BUB PRES	20.3208		23.7051 23.7051 23.7051 23.7009 23.7009
FAN PWR	1.9144		2.3926 2.3926 2.3926 2.3931 2.3931
THRUST	590.9226	00000000000000000000000000000000000000	565-4915 558-2554 551-1243 544-0486 540-6016
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THST PWR	986 985 985 985 985 986 986 986 986 986 986 986 986 986 986	######################################	33.4225 33.1031 32.8310
BUB PRES	233.7000 223.7000 223.70000 223.71000 223.71151 223.71000 200000 200000 200000 20000	20000000000000000000000000000000000000	26.0176 26.0115 26.0056
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FAN PWR		######################################
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80 48 TOT 10000000000000 mr nnnnnnnoopoopoopoopo 000400000400ru 58 3 2011-08041001-00801-01 11000-04700-000 KNOT 2 . . THS 25 α 0 0000 52 ш V@@@V@@@V@444 X -0----manamanamana 0 ш 000000000000 UB ANGL 25 NUNNNUNNUNNUN 7300 PITCH PWR t-JWWt+WtWtWWWWWWWWWWWW MOUND BUND BUND $\sigma\sigma\sigma\sigma\sigma\sigma\sigma\sigma\sigma\sigma\sigma\sigma\sigma\sigma$ AN SW 4 - N SIN 8004.0508 8003.0508 8001.52898 8001.52898 8001.52898 1994.18586 174682.1520 1746.16482 1746.1648 1756.6880 1756.8880 8. 2000 1. 00000 N POW 00 JUL V 00 **200000044400000** 01 rererentere BU V S THE шю URI 50 52 52 25 MA

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BUB PRES	20.3149 20.31888 20.31888 20.3188 20.3066 20.3066	21.7939 22.77939 22.77939 22.77939 22.77939 22.77939 22.77939 22.77939 22.77939 23.77939 23.77939 23.77939 24.77939 25.77939 27.7	233.70070 233.70070 233.66990 233.66990 233.66948 233.66948 233.66948 233.6948
FAN PWR	1.9147 1.9147 1.9147 1.9147 1.9148 1.9152 1.9151		72222222222222222222222222222222222222
THRUST	728.3645 724.0630 727.1658 712.7666 703.1736 692.9624 688.0237	7006. 7006. 7006. 7005. 70	6443. 6443. 6443. 6412. 6431. 6435. 6435. 6435. 6435. 6435. 6435. 6436.
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TOT PWR	49.683 49.1744 49.1744 49.1737 49.2486 49.55486 49.5554	18888888999 11161189450	44444444444444444444444444444444444444	888874890 886574890 76648890	46.3295 46.0017 45.6682 45.3335 45.1871 45.1677
THST PWR	47.2890 46.9391 46.7818 46.8564 47.1332 47.2065	79999999999999999999999999999999999999	44444444444444444444444444444444444444	44.745 66.0927 66.0927 66.144 67.164	43.5862 42.5562 42.9231 42.5873 42.5873 42.4415 42.4415
BUB PRES	23.6929 23.6929 23.7051 23.7051 23.7051 23.7090	44444444444444444444444444444444444444	2000 2000	44.8821 44.8821 44.8821 44.8821 44.8821 44.8821	26.0176 26.0056 26.0034 25.9995 25.9995 25.9995
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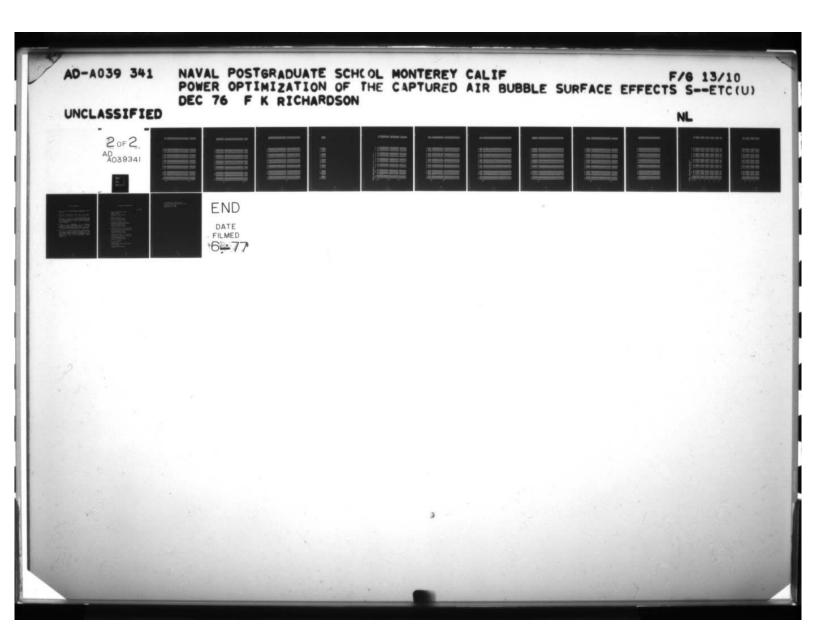
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TOT PWR	44.5816 44.5816 44.5816 44.5804 44.5085 44.5085	00000040444444444444444444444444444444	1.11.7 99.03.2 99.4038 99.6004
THST PWR	41.55229 41.55284 41.7078 41.7691 41.5516	α	55.00 56.00
BUB PRES	27.4485 27.44854 27.44824 27.4482 27.4402 27.4041	201010101010101010101010101010101010101	8.52 9.714 9.726 9.740 9.751
FAN PWR	2.9513 2.9513 2.95513 2.9526 2.8524 2.9569	######################################	0.0000000 0.00000000000000000000000000
THRUST	541.2400 541.8328 542.6401 543.6506 544.4497 541.6140	45000000000000000000000000000000000000	64.24 (64.647) 666.553 (68.388) 68.388 (69.127) 72.127
THETA	1.675 1.7233 1.865 2.1688 2.160	00000000000000000000000000000000000000	4 - 10000 4 - 10000 1 - 100000 1 - 1

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	TOT PWR	44.7950	79.52988 79.52988 79.15781 78.05908 77.55483 77.75681 76.25981 76.2597 76.2597 76.2597	77777777777777777777777777777777777777
5 KNOTS	THST PWR	42.1550	77. 77. 76. 76. 76. 76. 76. 76. 76. 76.	72 72 72 72 72 72 72 72 72 73 74 74 74 74 74 74 74 74 74 74 74 74 74
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FIGURE 6, T	THETA	7 2 3 1 2 800	00000000000000000000000000000000000000	77 70 70 70 70 70 70 70 70 70

OT PWR	7.9439	99999999999999999999999999999999999999	00000000000000000000000000000000000000
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FAN PWR	1.9169	00000000000000000000000000000000000000	14461164666911649444
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 THETA
 THRUST
 FAN PWR
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 47.1214

 1.6528
 509.2559
 3.3915
 29.5793
 42.9761
 46.3676

 1.8298
 501.6426
 3.3932
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 42.3336
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 1.9023
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 42.0247
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 2.0016
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000.2491 968.4595 967.12965 947.2895 947.9896 961.9896 961.9891 961.8891 961.8891 961.8891 9924-9922-6964 PWR TOT 200 33 母を見るとりももろも 85.8388 84.2388 82.7198 81.3735 80.6397 79.1916 77.7865 77.2336 966 Md KNOT THST 8888600PNW58 30.0 9749 9749 9790 9810 9829 9829 9768 9768 9768 9768 00000 887777 73398 13398 13398 13398 ES PR LLLLLLLLL 19. ------ANGLE 808 *นนนนนนนนนน* 0116555552 PITCH 7300 PWR 0000000000000 FAN -----NANNANANA SW POWER VS 050.1980 032.9075 014.5867 005.4446 996.5752 987.4178 573.1985 965.3130 949.2249 0000 7852 01852 9302 9302 9320 9320 9320 9324 THRUST 525. 8880-2000 8880-2000 968400-300 968400 BUBBLE FIGURE 7. THET/ 301 30

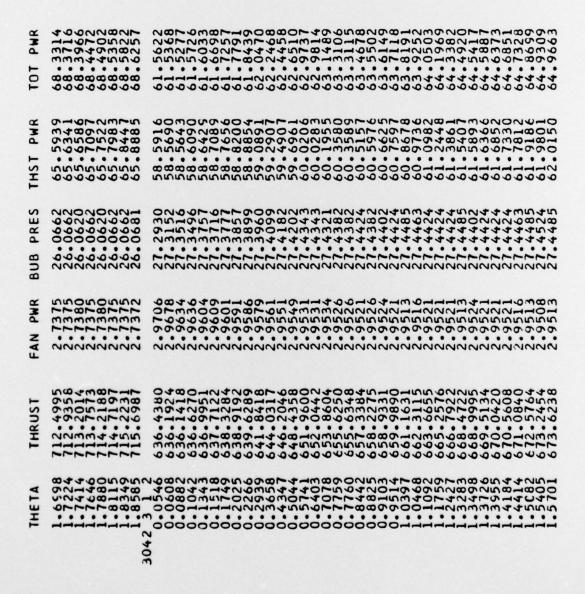
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FAN PWR	22.0950 20.095	240000000000000000000000000000000000000	20000000000000000000000000000000000000	04000000000000000000000000000000000000	00000000000000000000000000000000000000
THRUST	674-3474 675-1423 676-0508 676-0562 668-3833 659-2654	99991999999999999999999999999999999999	2551-110939 2551-110939 2551-110939 2551-110939	627-756 628-3756 629-629-629-630-630-6492 632-64	553 - 4016 553 - 4016 553 - 4016 553 - 4016 553 - 4016 553 - 4016 553 - 4016
THETA	604 604 602 1262 1264		1000 - 1000 1000 - 1000 1000 - 1000 1000 - 1000	110046 110020 10020	22229999999999999999999999999999999999

	TOT PWR	73.3135	34.0625 38.5167 47.3923 55.6044 63.2500 71.4922	33.9907 38.2027 46.7778 54.7513 62.1867 70.2121	33.9414 45.2164 52.5573 59.4598 66.9466	33. 93. 43. 673. 673. 66. 673. 66. 99. 99. 99. 99. 99. 99. 99. 99.	34.0018 36.2593 42.2419 48.2740 54.0522 60.4212	33.9852 35.6318 40.9183
	THST PWR	69.8135	32.3338 36.7879 45.6629 53.8749 61.5205	32.0767 36.2884 44.8624 52.8355 60.2706 68.2956	31.8226 43.08995 50.4367 51.3388 64.8287	31. 5475 34. 2821 40. 9751 47. 5664 53. 8134 60. 5999	31.4506 33.7066 39.6900 45.7218 51.5002 57.8685	31.2477 32.8936 38.1808
H ANGLE URE	BUB PRES	30.0000	19.4875 19.5156 19.6389 19.6631 19.6692 19.6692	20.3289 20.3289 20.3080 20.2905 20.2866 20.2866	21.8142 21.8020 21.7878 21.7859 21.7798 21.7798	23.7212 23.7151 23.7151 23.7192 23.7131 23.7131	24.8352 24.8230 24.8291 24.8269 24.8269 24.8269	26.0662 26.0601 26.0662
IST VS PITC	FAN PWR	3.5000	1.7287 1.7298 1.7295 1.7295 1.7295	1.9140 1.9143 1.9154 1.9158 1.9161	2.1198 2.1205 2.1206 2.1210 2.1210	2.3907 2.3914 2.3914 2.3917 2.3917 2.3917	22.5551 22.5551 22.5551 22.5551 25522 72.5551	2.7375 2.7382 2.7375
EED AND BU	THRUST	910.0000	421.4622 479.5212 595.2046 702.2454 801.9033	418.1118 473.0100 584.7698 688.6968 785.6116 890.2146	414.7998 460.8503 561.6655 657.4294 747.3962 844.9854	411.2136 446.8577 534.0994 620.0151 701.4436 789.9041	409.9500 439.3574 517.3491 595.9714 671.2920 754.3010	407.3052 428.7598 497.6772
FIGURE VARIABLE SP	THETA	50203010	00000000000000000000000000000000000000		22.60	22.22.28.48.48.69.89.89.89.89.89.89.89.89.89.89.89.89.89	22.1.25 22.1.25 22.1.25 22.1.30 22.1.30 22.1.30 23.1.3	2467 215 992

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TOT PWR	46.3860 51.7148 57.5980	33.7536 34.9880 39.4469 44.3210 49.1302 54.5357	33.1345 34.1153 38.1010 42.3958 46.7338 51.6120	32. 7528 32. 8963 36. 5044 40. 4713 44. 1246 47. 620
THST PWR	43.6478 48.9766 54.8605	30.7977 32.0349 36.4938 41.3686 46.1781 51.5844	29.9410 30.9275 34.9155 39.2110 43.5494	29.3636 29.5076 33.1195 37.0883 40.7398
BUB PRES	26.0601 26.0601 26.0662	27.412 27.44343 27.44402 27.4424 27.4424 27.4424	28.5723 28.7056 28.7561 28.7761 28.7761 28.7703	29.6357 29.6360 29.7346 29.7771 29.7368 29.6602
FAN PWR	2.7382 2.7382 2.7375	22.99531 22.99531 22.99531 22.99524 25.9521 25.9521	3.1935 3.1878 3.1858 3.1844 3.1844	200 200 200 200 200 200 200 200 200 200
THRUST	568.9382 638.3970 715.0918	401.4395 417.5667 475.6868 539.2288 601.9192 672.3894	390.2734 403.1318 455.1138 511.1057 567.6548 631.2336	382.7468 384.6240 431.7643 483.4358 531.0327 577.0994
THETA	1.889 1.848 1.835	22.00.11.00.00.00.00.00.00.00.00.00.00.00.	200000000000000000000000000000000000000	6 2 1 1 2 5 2 1 1 2 5 2 1 1 2 5 2 1 2 2 1 2 1 2 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1
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7. Lt. Frederick K. Richardson, USN
Strategic Systems Project Office, SP-273
Department of the Navy
Washington, D.C. 10091